

HW # VII

10.8

Assume $T_{air} = 25^\circ C$.

(a) $k_{glass} = 0.7 (W/m \cdot K)$ $k_{air} = 0.025 (W/m \cdot K)$

$Q = U A \Delta T$

$$U_1 = \frac{1}{2x \frac{L_{glass}}{k_{glass}} + \frac{L_{air}}{k_{air}}} = \frac{1}{2x \frac{0.0025}{0.7} + \frac{0.001}{0.025}} = \underline{21.2 (W/m^2 \cdot K)}$$

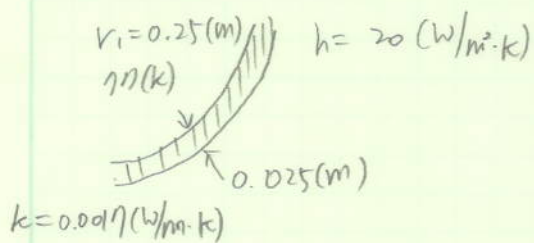
(b)

$$U_2 = \frac{(1 + 0.01)}{2x \frac{L_{glass}}{k_{glass}} + \frac{1}{h_{air}}} = \frac{1}{2x \frac{L_{glass}}{k_{glass}}}$$

$$\frac{1}{h_{air}} = 2x \frac{0.0025}{0.7} - \frac{1}{1.01} \times 2x \frac{0.0025}{0.7} = 0.000070721$$

$h_{air} = 14140 (W/m^2 \cdot K)$

10.24



$$\bar{r}_i = r_i + \Delta r \frac{1}{2} = 0.25 + 0.0125 = 0.2625 (m)$$

$$Q = 4\pi \bar{r}_i^2 U \Delta T$$

$$U = \frac{1}{\frac{1}{h} + \frac{\Delta r}{k}} = \frac{1}{\frac{1}{20} + \frac{0.025}{0.0017}} = 0.068 (W/m^2 \cdot K)$$

$$Q = 4\pi (0.2625)^2 (0.068) (300 - 17) = 13 (W)$$

$$\dot{m} = \frac{13 (W)}{2x 10^5 (J/kg)} = 0.000065 (kg/s) = \underline{5.6 (kg/day)}$$

$(day) = 86400 (sec)$

11.18

the copper body is "thick" (semi-infinite medium)

$$k_{\text{copper}} = 401 \text{ (W/m}\cdot\text{K)} \quad (\alpha_T)_{\text{copper}} = 117 \times 10^{-6} \text{ (m}^2\text{/s)}$$

From (eq. 11.3.20)

$$\bar{\epsilon} = \left(\frac{h}{k_r}\right)^2 \alpha_T t = \left(\frac{100}{401}\right)^2 \times 117 \times 10^{-6} t = 7.3 \times 10^{-6} t.$$

Get the surface temperature by using Fig 11.3.2 ($\theta - \bar{\epsilon}^{1/2}$).

t [s]	$\bar{\epsilon}$	$\theta (= \frac{T-30}{120-30})$	T [°C]
2000	0.0146	0.98	109.2
8000	0.0584	0.98	100.2
20000	0.146	0.62	91.2
100000	0.73	0.46	71.4
<u>10^6</u>	<u>7.3</u>	<u>0.06</u>	<u>35.4</u>

$$\underline{t = 10^6 \text{ (sec)}}$$

$$\delta_T = 4\sqrt{\alpha_T t} = 4\sqrt{117 \times 10^{-6} \times 10^6} = 43.27.$$

$$\text{thickness} = 2 \times \delta_T = \underline{86.54 \text{ (m)}}$$

$$11.29. \quad \theta_1 = A_1 e^{-\lambda_1^2 X_{F0}} \cos \lambda_1 \xi$$

$$\begin{aligned} \bar{\theta}_1 &= \int_0^1 \theta_1(\xi, t) d\xi = A_1 e^{-\lambda_1^2 X_{F0}} \int_0^1 \cos \lambda_1 \xi d\xi \\ &= \frac{A_1}{\lambda_1} \sin \lambda_1 e^{-\lambda_1^2 X_{F0}} \end{aligned}$$

For $B_1 = 0$, from Fig. 11.1.4(a)

$$A_1 = 1.24 \quad \text{and} \quad \lambda_1 / \sqrt{B} = 0.46.$$

$$\sin \lambda_1 = \sin 1.3 = 0.964$$

$$\bar{\theta}_1 = \frac{1.24}{1.3} (0.964) e^{-1.69 X_{F0}} = \underline{0.92 e^{-1.69 X_{F0}}}$$

$$12.7. \quad \theta = \frac{T_2 - T_R}{T_1 - T_R} = \exp(-4 \cdot St \cdot \frac{L}{D})$$

$$St = \frac{\bar{Nu}}{Re \cdot Pr}$$

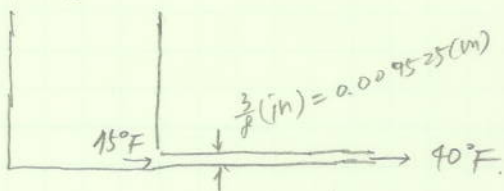
$$Re \cdot Pr \cdot \frac{D}{L} = \frac{40}{\pi} (6) \frac{1}{60} = \frac{4}{\pi} \quad \bar{Nu} = 3.66 \quad (\text{from Fig 12.5.6})$$

$$St = \frac{3.66}{\frac{40}{\pi} (6)} = 0.048$$

$$\theta = \exp(-4 \times 0.048 \times 60) = e^{-11.5} = 10^{-5}$$

$$T_2 = 90 + (50 - 90) \times 10^{-5} \approx \underline{90^\circ\text{C}}$$

12.62.



$$T_1 = 15^\circ\text{F} \quad T_2 = 40^\circ\text{F}$$

$$\dot{Q} = 2 \text{ (gal/min)} = \frac{0.001571}{60} \text{ (m}^3\text{/s)} \quad Pr = 8.09$$

$$\rho = 1000 \text{ (kg/m}^3) \quad \mu = 1.138 \times 10^{-3} \text{ (kg/m}\cdot\text{s)}$$

$$Re = \frac{4 \dot{Q}}{\pi \cdot D \cdot \mu} = \frac{4 \times 1000 \times 0.001571 / 60}{\pi \times 0.009525 \times 1.138 \times 10^{-3}} = 14822 \text{ (turbulent)}$$

$$f = 0.08 Re^{-0.25} = 0.00725 \quad j_H = f/2 = 0.0036 \quad Pr^{1/3} = 4.03$$

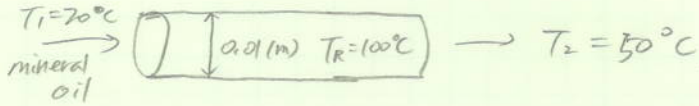
$$St = \frac{j_H}{Pr^{1/3}} = 0.0008933$$

$$\theta = \frac{T_2 - T_R}{T_1 - T_R} = \exp(-4 \cdot St \cdot \frac{L}{D})$$

$$= \frac{40 - 32}{15 - 32} = \exp(-4 \times 0.0008933 \times \frac{L}{0.009525}) = 0.186$$

$$\Rightarrow L = -\frac{1}{4} \times \frac{1}{0.0008933} \times 0.009525 \times \ln(0.186) = \underline{4.67 \text{ (m)}}$$

13.12



$$\dot{Q} = 0.01 (\text{J/s})$$

$$c_p = 2000 (\text{J/kg}\cdot\text{K})$$

$$k = 0.147 (\text{W/m}\cdot\text{K})$$

$$\rho = 830 (\text{kg/m}^3)$$

the mean of the wall temperature and the oil temperature: $T_f = \frac{1}{2}(35 + 100) = 67.5^\circ\text{C}$.

$$\text{At } 67.5^\circ\text{C}, \mu = 0.016 \exp(-0.0384 T) = 0.0058 (\text{kg/m}\cdot\text{s})$$

$$\theta = \frac{T_2 - T_R}{T_i - T_R} = \frac{50 - 100}{20 - 100} = 0.625$$

$$Pr = \frac{\mu c_p}{k} = \frac{0.0058 \times 2000}{0.147} = 79$$

$$Re = \frac{4\dot{Q}}{\pi \cdot D \cdot \mu} = \frac{4 \times 0.00001 \times 830}{\pi \times 0.01 \times 0.0058} = 182$$

$$z^* = L^* = \frac{\alpha L}{LR^2} = \frac{4L/D}{Pr \cdot Re} \approx 0.09 \Rightarrow \frac{L}{D} = \frac{0.09 Re \cdot Pr}{4} = 321$$

$$L = 321 \times 0.01 = 3.21 (\text{m}) \quad \therefore \text{a rough estimate}$$

$$\overline{Nu}_D = 1.6 (Re \cdot Pr \cdot D/L)^{1/3} \quad (\text{eq. 12.5-49})$$

$$\overline{St} \cdot \frac{L}{D} = \frac{\overline{Nu}}{Re \cdot Pr \cdot D/L} = 1.6 (Re \cdot Pr \cdot D/L)^{-2/3}$$

$$\theta = 0.625 = \exp[-4 \cdot 1.6 (Re \cdot Pr \cdot D/L)^{-2/3}] \Rightarrow Re \cdot Pr \cdot D/L = 50$$

$$L/D = Re \cdot Pr / 50 = 286 \quad \underline{L = 2.86 (\text{m})}$$

$$\overline{h} = \frac{k \cdot \overline{Nu}}{D} = \frac{0.147 (5.9)}{0.01} = \underline{86.5 (\text{W/m}^2\cdot\text{K})}$$

$$13.23. \quad 4 \cdot \overline{St} \cdot \frac{L}{D} = \frac{\pi D L \cdot \overline{h}}{\dot{m} \cdot c_p} = \frac{h A}{\dot{m} \cdot c_p} \equiv \frac{UA}{\dot{m} \cdot c_p}$$

$$\Rightarrow \underline{\overline{St} \equiv \frac{D}{4L} \frac{UA}{\dot{m} \cdot c_p} = \frac{D}{4L} \text{NTU}}$$