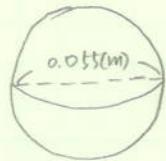


15.2



$$\epsilon = 0.95 \quad Q = 15 \text{ (W)} \quad A = 4\pi \left(\frac{0.055}{2}\right)^2 = 9.5 \times 10^{-3} \text{ (m}^2\text{)}$$

$$T_{\text{air}} = 25^\circ\text{C} \quad T_{\text{surface}} = ?$$

$$\dot{q}_{\text{tot}} = \frac{Q}{A} = \dot{q}_{\text{rad}} + \dot{q}_{\text{conv}}$$

* radiation

$$\dot{q}_{\text{rad}} = \epsilon \sigma T_s^4 = 0.95 (5.67 \times 10^{-8}) T_s^4$$

* convection

Assume T_s for evaluation of physical properties.

$$T_s = 200^\circ\text{C} \quad T_f = \frac{1}{2}(200+20) = 110^\circ\text{C}$$

$$\gamma = 4 \times 10^7 \text{ (m}^3\text{.K}^{-1}\text{)} \quad Pr = 0.7 \quad \mu = 2.1 \times 10^{-5} \quad \rho = 0.84 \quad k_T = 0.03$$

$$\nu = 2.5 \times 10^{-5} \text{ (m}^2\text{/s)}$$

$$Gr = \frac{\gamma \beta L^3 \Delta T}{\nu^2} = \frac{9.8 \times \left(\frac{1}{3.83}\right) (0.055/2)^3 (T_s - 293)}{(2.5 \times 10^{-5})^2} = 851 (T_s - 293)$$

$$Ra = Gr \cdot Pr = 851 (T_s - 293) \times 0.7 = 596 (T_s - 293)$$

$$Nu = 0.53 Ra^{0.25} = \frac{h_{\text{conv}}}{k_T} \cdot L$$

$$\Rightarrow h_{\text{conv}} = \frac{k_T}{L} Nu = \frac{0.03}{0.055/2} \times (0.53) \times (596)^{0.25} (T_s - 293)^{0.25} = 2.8 (T_s - 293)^{0.25}$$

$$\dot{q}_{\text{total}} = \dot{q}_{\text{rad}} + \dot{q}_{\text{conv}} = 0.95 (5.67 \times 10^{-8}) T_s^4 + 2.8 (T_s - 293)^{0.25} (T_s - 293) = \frac{15}{9.5 \times 10^{-3}}$$

$$= 17895$$

Through a rough trial & error calculation,

we can get $\underline{T_s = 511.5 \text{ (K)}} = 302^\circ\text{C}$.* For $T_f = \frac{1}{2}(302+20) = 161^\circ\text{C}$,

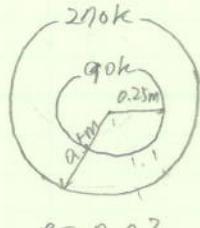
$$\gamma = 2 \times 10^7 \quad \mu = 2.3 \times 10^{-5} \quad \rho = 0.74 \quad Pr = 0.7 \quad k_T = 0.034 \quad \nu = 3.1 \times 10^{-5}$$

$$Gr = \frac{(9.8) (1/434)}{(3.1 \times 10^{-5})^2} \left(\frac{0.055}{2} \right)^3 (T_s - 293) = 511 (T_s - 293)$$

$$Ra = 511 (T_s - 293) \times 0.7 = 362 (T_s - 293)$$

$$h_{\text{conv}} = \frac{0.034}{0.028} (0.53) (362)^{0.25} (T_s - 293)^{0.25} = 2.8 (T_s - 293)^{0.25} \Rightarrow T_s \text{ is changed from } 200^\circ\text{C to } 302^\circ\text{C, but } h_{\text{conv}} \text{ is unchanged!!}$$

15.4.



$$e = 0.03$$

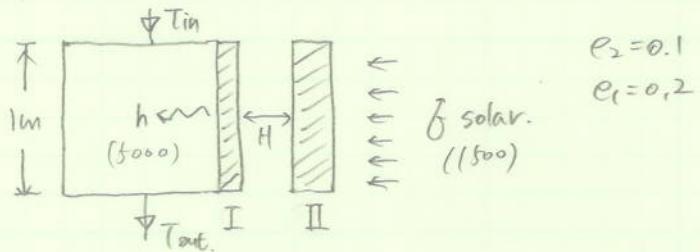
$$h = 2 \times 10^8 \text{ (J/kg)}$$

$$\begin{aligned} \bar{q}_{12} &= \frac{\sigma(T_1^4 - T_2^4)}{1 + \left(\frac{1}{e_1} - 1\right) + \frac{1}{e_2} \left(\frac{1}{e_2} - 1\right)} = \frac{\sigma(T_1^4 - T_2^4)}{\frac{1.25}{e} - 0.25} \\ &= \frac{5.67 \times 10^{-8} (90^4 - 270^4)}{\frac{1.25}{0.03} - 0.25} = 7.2 \text{ (W/m}^2\text{)} \end{aligned}$$

$$\bar{q}_{12} = \frac{Q_{12}}{A} \Rightarrow Q_{12} = \bar{q}_{12} \cdot A = \bar{q}_{12} \cdot 4\pi(r_1)^2 = 7.2 \times 4\pi \times 0.25^2 = 5.65 \text{ (W)}$$

$$\dot{m} = \frac{Q}{2 \times 10^5} = \frac{5.65}{2 \times 10^5} = 2.825 \times 10^{-5} \text{ (kg/s)}$$

15.7.



$$\text{i) Plate I: } \bar{q}_{\text{solar}} \cdot A = \bar{q}_{21} + e_2 \sigma T_2^4$$

$$1500(0.1) = \bar{q}_{21} + 0.1 \times (5.67 \times 10^{-8}) T_2^4$$

$$\Rightarrow 150 = \bar{q}_{21} + 5.7 \times 10^{-9} T_2^4$$

$$\text{ii) Plate II: } \bar{q}_{21} = h(T_1 - T_{\text{out}}) \quad (\text{The heat transfer fluid is well-mixed. and at the uniform temperature } T_{\text{out}})$$

$$\bar{q}_{21} = 5000(T_1 - T_{\text{out}})$$

$$\text{iii) Fluid: } \omega C_p(T_{\text{out}} - T_{\text{in}}) = \bar{q}_{21} \cdot A_1 \quad (\text{Assume } T_{\text{in}} = 300 \text{ (K)})$$

$$\frac{100(4000)}{3600} (T_{\text{out}} - 300) = \frac{\pi}{4} \bar{q}_{21}$$

$$\Rightarrow \bar{q}_{21} = 141(T_{\text{out}} - 300)$$

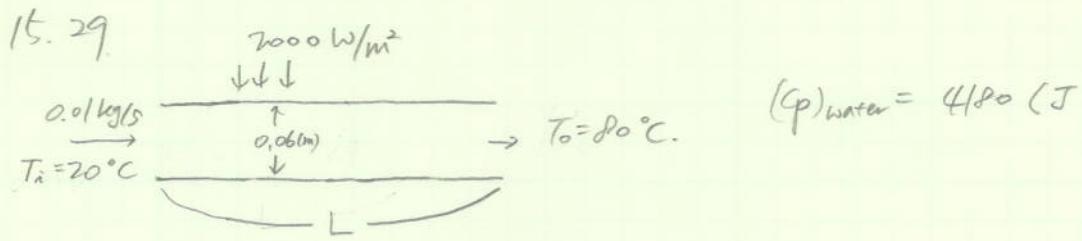
$$\text{iv) Eq 15.2.5: } \bar{q}_{21} = \frac{\sigma(T_2^4 - T_1^4)}{1/e_1 + 1/e_2 - 2 + \frac{2}{H F_{12}}} = \frac{5.67 \times 10^{-8}(T_2^4 - T_1^4)}{13 + \frac{2}{H F_{12}}} \approx 3.9 \times 10^{-9}(T_2^4 - T_1^4)$$

* From Fig 15.2.5 $(F_{12})_{\max} = 0.72$ $D/H = 6.5$ for disks
 $(F_{12})_{\min} = 0.07$ $D/H = 0.5$ for disks

Changes in H (or D/H) has no significant effect on q_{21} and results.

From 4 equations (plate I, II, Fluid, eq. 15.2.31), we can get q_{21} , T_1 , T_2 , T_{out} .

$$q_{21} = 43 \text{ (W/m}^2\text{)} \quad T_1 = 300.3 \text{ (K)} \quad T_2 = 310.5 \text{ (K)} \quad T_{out} = 300.3 \text{ (K)}$$



$$(a) \dot{m} C_p \Delta T = \pi D L q_{abs}$$

$$L = \frac{\dot{m} C_p \Delta T}{\pi D \cdot q_{abs}} = \frac{0.01(4180) \cdot 60}{\pi(0.06) 2000} = 6.65 \text{ cm}$$

(b) At the end of the tube

$$q = h(T_s - T_o) \quad \text{To find } h, \text{ begin with } Re = \frac{4L}{\pi D \cdot u}$$

For the evaluation of properties, guess $T_s = 100^\circ\text{C}$.

$$T_f = \frac{1}{2}(100 + 80) = 90^\circ\text{C} \Rightarrow \mu = 0.3 \times 10^{-3} \quad k_r = 0.7 \quad Pr = 2.$$

$$Re = \frac{4 \times 0.01}{\pi(0.06)(0.3 \times 10^{-3})} = 707$$

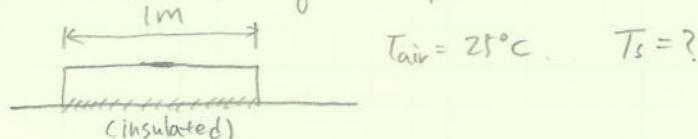
$$\frac{L}{D \cdot Re \cdot Pr} = \frac{6.65}{0.06(707)^2} = 0.08 \Rightarrow \frac{2L}{D \cdot Re \cdot Pr} = 0.16.$$

From Fig 12.5.5 $Nu_D \approx 4.4$

$$h = \frac{k_r \cdot Nu}{D} = \frac{0.7(4.4)}{0.06} = 51 \text{ (W/m}^2\text{K)}$$

$$T_s - T_o = \frac{q}{h} = \frac{2000}{51} \Rightarrow T_s = 80 + 39 = 119^\circ\text{C}$$

15. 34. $\downarrow \downarrow \downarrow \downarrow \downarrow \downarrow$ $q = 1000 \text{ W/m}^2$.



Assume black body behavior.

$$q_{solar} = \sigma T_s^4 + h_{conv} (T_s - T_a)$$

$$\text{guess } T_s = 100^\circ\text{C} \quad T_f = \frac{1}{2}(100 + 25) = 62.5^\circ\text{C}$$

$$\Rightarrow \gamma = 8 \times 10^7 (\text{m}^{-3}\text{K}^{-4}) \quad Pr = 0.72$$

$$Ra = \gamma \cdot Pr \cdot L^3 \cdot \Delta T = 8 \times 10^7 (0.72) (1)^3 (100 - 25) = 3.1 \times 10^9$$

From Table 14.4.1, $Nu_r = 0.15(Ra)^{0.33} = 204$

$$h = Nu \frac{k_{air}}{L} = 204 \frac{(0.028)}{0.9} = 6.3 \text{ (W/m}^2\text{.K)}$$

$$h\Delta T = 6.3(15) = 94.5$$

$$\dot{q}_f = 1000 = 5.67 \times 10^{-8} \cdot Ts^4 + 477 \Rightarrow Ts = 310 \text{ K} = 37^\circ C$$

change to $T_f = \frac{1}{2}(37+25) = 31^\circ C$. and continue by trial & error

$T_s = 69^\circ C$