Solution VII

1 Since the tube is long, assume that the flow is fully developed. We apply a pressure difference between the two ends so the Navier–Stoke equation along the pipe takes the two forms

$$0 = -p_x + \frac{\mu_L}{r} \frac{\mathrm{d}}{\mathrm{d}r} \left(r \frac{\mathrm{d}u_W}{\mathrm{d}r} \right), \qquad 0 = -p_x + \frac{\mu_A}{r} \frac{\mathrm{d}}{\mathrm{d}r} \left(r \frac{\mathrm{d}u_A}{\mathrm{d}r} \right).$$

We can integrate these two equations to get

$$u_L = \frac{p_x}{4\mu_L}r^2 + A + B\log r, \qquad u_A = \frac{p_x}{4\mu_A}r^2 + C + D\log r.$$

The boundary conditions at the pipe walls are $u_L = 0$ at r = a. At the interface between air and liquid at r = h, the velocity, u, and the shear stress, $\mu du/dr$, are continuous. (The thickness of the liquid layer is a - h.) In addition the logarithmic term must vanish along the centerline of the pipe so D = 0. The remaining three conditions are

$$\frac{p_x}{4\mu_L}a^2 + A + B\log a = 0, \quad \frac{p_x}{4\mu_L}h^2 + A + B\log h = \frac{p_x}{4\mu_A}h^2 + C, \quad \frac{p_x}{2}h + \frac{\mu_L B}{h} = \frac{p_x}{2}h.$$

Note that in the shear stress relation, the pressure terms cancel. Hence B = 0 and we obtain

$$u_L = \frac{p_x}{4\mu_L}(r^2 - a^2), \qquad u_A = \frac{p_x}{4\mu_A}(r^2 - h^2) + \frac{p_x}{4\mu_L}(h^2 - a^2).$$

The volume flux in the liquid is

$$Q_L = \int_h^a u_L 2\pi r \, \mathrm{d}r = \frac{\pi p_x}{4\mu_L} (a^2 - h^2)^2$$

while the volume flux in the air is

$$Q_A = \int_0^a u_A 2\pi r \, \mathrm{d}r = \frac{\pi p_x}{4\mu_A} h^2 \left[h^2 + \frac{2\mu_A}{\mu_L} (a^2 - h^2) \right].$$

Their ratio is hence

$$\frac{Q_A}{Q_L} = \frac{\mu_L}{\mu_A} \frac{1 + 2(\mu_A/\mu_L)(a^2/h^2 - 1)}{(1 - a^2/h^2)^2}.$$

Note that this does not depend on the pressure gradient, just on the ratio of viscosities μ_L/μ_A and the geometric ratio h/a.

2 Take the *x*-axis along the slope and the *z*-axis perpendicular to it. The flow is steady $(\partial_t = 0)$ and fully-developed $(\partial_x = 0)$. Then the *x*-momentum equations in the two layers become

$$0 = \frac{\mathrm{d}^2 u_{T,B}}{\mathrm{d}z^2} - \frac{\rho g}{\mu_{T,B}}\sin\theta.$$

The boundary conditions are no slip at the bottom: $u_B = 0$ at z = 0, no stress at the top of the upper layer (neglecting shear stress in the air): $\mu_T du_T/dz = 0$ at $z = h_B + h_T$, and continuity of velocity and stress at the interface: $u_B = u_T$ and $\mu_B du_B/dz = \mu_T du_T/dz$ at $z = h_B$. Integrate and solve for the four constants using the four boundary and interfacial conditions:

$$u_B = \frac{\rho g}{2\mu_B} \sin \theta [z^2 - 2(h_T + h_B)z],$$

$$u_T = \frac{\rho g}{2\mu_T} g \sin \theta [z^2 - 2(h_T + h_B)z + h_B(2h_T + h_B)(1 - \mu_T/\mu_B)]$$

The velocity in the bottom layer does not depend on μ_T because the shear stress at the top of the bottom layer is

$$\mu_B \mathrm{d} u_B / \mathrm{d} z = -\rho g \sin \theta h_T,$$

which is determined only by the weight of the fluid in the top layer.

3 As in class, look for a solution of the form

$$u_{\theta} = \frac{\Gamma}{2\pi r} f(\eta),$$

where $\eta = r^2/4\nu t$. We obtain the same equation as in class, f'' + f' = 0, but with different boundary conditions. Now at t = 0 and as $r \to \infty$, $f \to 0$ (no initial vortex and no flow at infinity), while at r = 0, f = 1 (vortex now in flow). Hence $f = e^{-\eta}$. This gives the solution.

4 Initial conditions: u(y,0) = 0. Boundary conditions: u(-h,t) = u(h,t) = 0. We write the velocity as the steady-state solution that we expect to see for large time plus a Fourier cosine series that satisfies the boundary conditions:

$$u(y,t) = -\frac{P}{\mu}(y^2 - h^2) + \sum_{n=0}^{\infty} a_n(t) \cos\left[n + \frac{1}{2}\right) \frac{\pi y}{h}.$$

Substituting this into the governing equation and cancelling off the pressure term, multiplying by $\cos\left[(m + \frac{1}{2})\pi y/h\right]$ and integrating over the channel gives

$$\frac{\mathrm{d}a_m}{\mathrm{d}t} = -\frac{\nu(m+\frac{1}{2})^2 \pi^2}{h^2} a_m$$

The solution to this set of equations is

$$a_n(t) = a_m(0) e^{-\mu[(n+\frac{1}{2})\pi/2h]^2 t}$$

It now remains to find $a_m(0)$ from the equation

$$0 = -\frac{P}{\mu}(y^2 - h^2) + \sum_{n=1}^{\infty} a_n(0) \cos\left[(n + \frac{1}{2})\pi y/h\right].$$

Once again multiply by $\cos\left[(m+\frac{1}{2})\pi y/h\right]$ and integrate to get

$$a_m(0) = \frac{P}{\mu h} \int_{-h}^{h} \cos\left[(m + \frac{1}{2})\pi y/h(y^2 - h^2) \,\mathrm{d}y = -\frac{32(-1)^n h^2}{(2m+1)^3 \pi^3} \frac{P}{\mu}\right]$$

Velocity profiles are shown in Figure 1.



Figure 1: Temporal development of Poiseuille flow in channel.